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# 初始滑面正应力假定对一般三维边坡稳定性影响

刘华丽<sup>1</sup>, 朱大勇<sup>2</sup>

(1. 陆军工程大学, 江苏 南京 210007; 2. 浙江大学宁波理工学院 土木建筑工程学院, 浙江 宁波 315100)

**摘要:** 基于滑面正应力修正的边坡安全系数显示解法,研究滑面初始正应力分布对边坡稳定性的影响。假设滑面的初始正应力分布,用含有待定参数的修正函数对其进行修正,使滑体满足所有方向的力和力矩平衡条件,推导出满足所有平衡条件的安全系数方程,得到边坡安全系数值,对边坡稳定性进行评判分析。结果表明,对于三维对称边坡,不同初始滑面正应力分布对边坡稳定性影响较小,安全系数最大差别控制在 5% 以内;对于一般三维形状边坡,不同初始正应力分布对边坡稳定性影响较大,三维安全系数的差别在 6.2%,与其他方法的差别最大达到 28.7%,需要进一步验证滑面正应力分布的合理性才能应用于工程实践。

**关键词:** 滑面初始正应力; 安全系数; 边坡稳定; 影响

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## Influence of Initial Normal Stress of Slip on the Stability of Common Three-dimensional Slopes

LIU Huali<sup>1</sup>, ZHU Dayong<sup>2</sup>

(1. Army Engineering University of PLA, Nanjing 210007, Jiangsu, China;

2. School of Civil Engineering and Architecture, Ningbo Institute of Technology, Zhejiang University, Ningbo 315100, Zhejiang, China)

**Abstract:** The influence of the distribution of the initial normal stress on the stability of slope was investigated using the explicit solution of slope safety factor based on modification of normal stress distribution over the slip surface. The initial normal stress distribution over the slip surface was assumed and then modified by a correction function with undetermined parameters so the slip mass can satisfy the equilibrium conditions of force and moment in all directions. An equation satisfying all the equilibrium conditions was derived, and the safety factor was calculated. Slope stability was also evaluated. Computation results show the minimal influence of different distributions of initial normal stress on the 3D symmetrical slope. The maximum difference in the safety factor is below 5%. For the common slope, the influence is relatively large and the difference in the safety factor is 6.2%. The maximum difference compared with the other methods is 28.7%. Therefore, verifying the rationality of normal stress distribution over the sliding surface is necessary in engineering practice.

**Keywords:** initial normal stress of slip; safety factor; slope stability; influence

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第一作者简介:刘华丽(1975—),女,博士,副教授,主要从事岩土工程学习与研究。E-mail:1429211066@qq.com。

## 0 引言

极限平衡法是边坡稳定分析中最常用的方法,专家认为只有严格满足力和力矩平衡条件,得出的安全系数才可以满足工程需要,严格满足力和力矩平衡条件求出的安全系数称为严格法安全系数<sup>[1]</sup>。但研究表明严格法安全系数差别是很明显的,而且不一定能给出唯一的安全系数值。一些方法为了使问题静定可解,引入了大量的假定,尤其是三维边坡<sup>[2-3]</sup>,导致不同方法之间安全系数相差很大,使三维严格安全系数方法在工程中无法应用。研究发现,对于任意形状二维边坡,安全系数的差别在15%以内,对于三维边坡,安全系数的差别达到40%,而且这些方法无法获取严格的3D极限平衡解<sup>[4]</sup>。近几年来的研究表明,在2D分析中不划分条块,直接调整滑面上的正应力分布,可以得到满足严格整体平衡条件的安全系数显示解,同样在三维稳定性分析中,不划分条柱,调整滑体空间滑面上正应力分布,可以求出满足所有平衡条件的安全系数,称为三维安全系数显式解法<sup>[5]</sup>。安全系数显式解法首先假定滑面初始正应力分布,然后用含有待定系数参数修正滑面初始正应力,推导出满足所有方向上的力和力矩平衡方程,采用数学方法求出边坡的安全系数,判断边坡稳定性状况。安全系数的求解过程中,初始滑面正应力假定是否影响边坡安全系数非常关键。影响过大,无法满足工程要求。若无影响,则可很方便地应用于工程实践中。

## 1 基本假定与平衡方程

用函数 $g(x, y)$ 和 $s(x, y)$ 描述边坡滑体水平面和滑动面。用 $W$ 表示滑体的总体重,用 $K_c W$ 表示水平方向的地震力,其中 $K_c$ 为地震影响系数。用 $\sigma(x, y)$ 和 $\tau(x, y)$ 分别表示作用于滑面上的正应力与剪切力,用 $U(x, y)$ 表示滑体的水压力(图1)。

假设滑面正应力为:

$$\sigma(x, y) = \sigma_0(x, y) \cdot [\lambda_1 x + \lambda_2 y + \lambda_3 x^2 + \lambda_4 y^2 + \lambda_5] \quad (1)$$

图2为条柱受力示意图。沿 $x$ 轴和 $y$ 轴分别取第 $j$ 个和第 $i$ 个条柱。 $\sigma(x, y), \tau(x, y)$ 分别为条柱底面的滑面法向力和剪应力。设法向力的方向余弦为 $(n_x, n_y, n_z)$ ,剪切力的方向余弦为 $(m_x, m_y, m_z)$ 。设条柱的中心点坐标为 $(x_c, y_c, z_c)$ , $\alpha$ 为条柱底面的倾角。设滑体在 $xoz$ 平面内可产生滑动,沿 $y$ 轴方向不产生滑动,则可得到 $m_y = 0$ 。

为简单起见,‘ $(x, y)$ ’在下文中可省略不写。

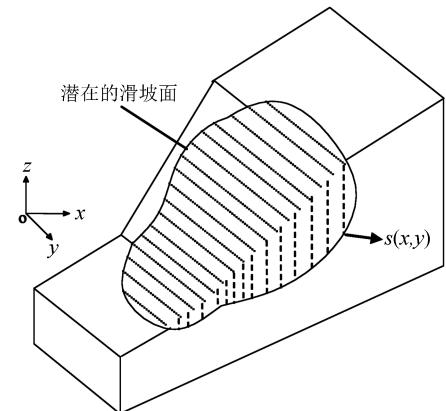


图1 三维滑面及相应坐标系

Fig.1 The 3D slip surface and coordinate

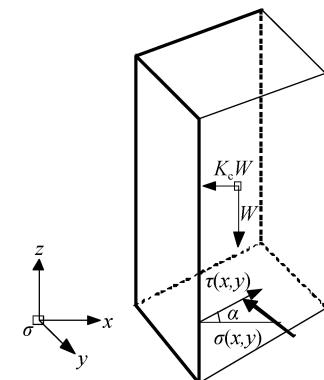


图2 条柱受力图

Fig.2 Forces acting on a column

根据外法线的定义,滑面外法线的方向数为 $\left(\frac{\partial s}{\partial x}, \frac{\partial s}{\partial y}, -1\right)$ 。由于滑面法向力方向与滑面外法线的方向相反,则滑面法向力的方向余弦为:

$$(n_x, n_y, n_z) = \left( -\frac{\frac{\partial s}{\partial x}}{\Delta}, -\frac{\frac{\partial s}{\partial y}}{\Delta}, \frac{1}{\Delta} \right), \quad (2)$$

$$\Delta = \sqrt{1 + \left(\frac{\partial s}{\partial x}\right)^2 + \left(\frac{\partial s}{\partial y}\right)^2}$$

剪切力方向与法向力方向是垂直关系,所以

$$(m_x, m_y, m_z) = \left( \frac{1}{\Delta'}, \frac{\frac{\partial s}{\partial x}}{\Delta'}, 0 \right), \quad \Delta' = \sqrt{1 + \left(\frac{\partial s}{\partial x}\right)^2} \quad (3)$$

设小条柱底面在 $xoy$ 平面上投影为长方形,面积为 $dx dy$ ,条柱底面的面积为 $dA$ 。则

$$dA = \frac{dx dy}{n_z} = \Delta dx dy = \sqrt{1 + \left(\frac{\partial s}{\partial x}\right)^2 + \left(\frac{\partial s}{\partial y}\right)^2} dx dy \quad (4)$$

三维滑体沿 $x, y, z$ 三个方向的力平衡和绕 $x, y, z$ 三个方向的力矩平衡方程分别为:

$$\begin{cases}
 \iint (\sigma \cdot dA \cdot n_x + \tau \cdot dA \cdot m_x) dx dy = \iint K_c \cdot w dx dy \\
 \iint (\sigma \cdot dA \cdot n_y + \tau \cdot dA \cdot m_y) dx dy = 0 \\
 \iint (\sigma \cdot dA \cdot n_z + \tau \cdot dA \cdot m_z) dx dy = \iint w dx dy \\
 \iint \sigma \cdot dA \cdot n_y \cdot s dx dy + \iint \tau \cdot dA \cdot m_y \cdot s dx dy - \iint \sigma \cdot dA \cdot n_z \cdot y dx dy - \iint \tau \cdot dA \cdot m_z \cdot y dx dy + \\
 \quad \iint w \cdot y_c dx dy = 0 \\
 - \iint \sigma \cdot dA \cdot n_x \cdot s dx dy - \iint \tau \cdot dA \cdot m_x \cdot s dx dy + \iint K_c w \cdot z_c dx dy + \iint \sigma \cdot dA \cdot n_z \cdot x dx dy + \\
 \quad \iint \tau \cdot dA \cdot m_z \cdot x dx dy - \iint w \cdot x_c dx dy = 0 \\
 \iint \sigma \cdot dA \cdot n_x \cdot y dx dy + \iint \tau \cdot dA \cdot m_x \cdot y dx dy - \iint K_c w \cdot y_c dx dy - \iint \sigma \cdot dA \cdot n_y \cdot x dx dy - \\
 \quad \iint \tau \cdot dA \cdot m_y \cdot x dx dy = 0
 \end{cases} \quad (5)$$

把式(2)、式(3)、式(4)式带入式(5)可得到

$$\begin{cases}
 - \iint \sigma \cdot \frac{\partial s}{\partial x} \cdot dx dy + \iint \tau \cdot \frac{\Delta}{\Delta'} dx dy = \iint K_c \cdot w dx dy \\
 - \iint \sigma \cdot \frac{\partial s}{\partial y} \cdot dx dy = 0 \\
 \iint \sigma \cdot dx dy + \iint \tau \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} dx dy = \iint w dx dy \\
 - \iint \sigma \cdot \frac{\partial s}{\partial y} \cdot s dx dy - \iint \sigma \cdot y \cdot dx dy - \iint \tau \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot y \cdot dx dy + \iint w \cdot y_c dx dy = 0 \\
 \iint \sigma \cdot \frac{\partial s}{\partial x} \cdot s dx dy - \iint \tau \cdot \frac{\Delta}{\Delta'} \cdot s dx dy + \iint K_c w \cdot z_c dx dy + \iint \sigma \cdot x \cdot dx dy + \\
 \quad \iint \tau \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot x \cdot dx dy - \iint w \cdot x_c dx dy = 0 \\
 - \iint \sigma \cdot \frac{\partial s}{\partial x} \cdot y \cdot dx dy + \iint \tau \cdot \frac{\Delta}{\Delta'} \cdot y \cdot dx dy - \iint K_c w \cdot y_c dx dy + \iint \sigma \cdot \frac{\partial s}{\partial y} \cdot x dx dy = 0
 \end{cases} \quad (6)$$

根据摩尔-库仑准则,

$$\tau = \frac{[\sigma - u] \cdot \tan\varphi + c}{F_s} \quad (7)$$

式中:  $\phi$  和  $c$  分别表示滑面有效内摩擦角和凝聚力。

以下用“ $\psi$ ”表示  $\tan\phi$ 。

把式(7)代入式(6)可得到

$$\begin{cases}
 \iint \left[ -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right] \sigma \cdot dx dy = \iint K_c w dx dy + \iint \frac{u \cdot \psi - c}{F_s} \frac{\Delta}{\Delta'} dx dy \\
 - \iint \frac{\partial s}{\partial y} \cdot \sigma \cdot dx dy = 0 \\
 \iint \left[ 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right] \sigma \cdot dx dy = \iint w dx dy + \iint \frac{u \cdot \psi - c}{F_s} \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} dx dy \\
 \iint \left[ -\frac{\partial s}{\partial y} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \cdot y \right] \sigma \cdot dx dy = - \iint w \cdot y_c dx dy - \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} y dx dy \\
 \iint \left[ \frac{\partial s}{\partial x} \cdot s - \frac{\psi \cdot s}{F_s} \frac{\Delta}{\Delta'} + x + \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \cdot x \right] \sigma \cdot dx dy = - \iint K_c w \cdot z_c dx dy + \iint w \cdot x_c dx dy - \\
 \quad \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\Delta}{\Delta'} s dx dy + \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot x dx dy \\
 \iint \left[ -\frac{\partial s}{\partial x} \cdot y + \frac{\Delta}{\Delta'} \frac{\psi \cdot y}{F_s} + \frac{\partial s}{\partial y} \cdot x \right] \cdot \sigma \cdot dx dy = \iint K_c w \cdot y_c dx dy + \iint \frac{u \cdot \psi - c}{F_s} \frac{\Delta}{\Delta'} \cdot y dx dy
 \end{cases} \quad (8)$$

把式(1)代入式(8)得

$$\begin{aligned}
 & \lambda_1 \iint x \cdot \left( -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \lambda_2 \iint y \cdot \left( -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_3 \iint x^2 \cdot \left( -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \lambda_4 \iint y^2 \cdot \left( -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_5 \iint \left( -\frac{\partial s}{\partial x} + \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy = \iint K_c w dx dy + \iint \frac{u \cdot \psi - c}{F_s} \frac{\Delta}{\Delta'} dx dy \\
 & - \lambda_1 \iint x \cdot \frac{\partial s}{\partial y} \cdot \sigma_0 dx dy - \lambda_2 \iint y \cdot \frac{\partial s}{\partial y} \cdot \sigma_0 dx dy - \lambda_3 \iint x^2 \cdot \frac{\partial s}{\partial y} \cdot \sigma_0 dx dy - \lambda_4 \iint y^2 \cdot \frac{\partial s}{\partial y} \cdot \sigma_0 dx dy - \\
 & \lambda_5 \iint \frac{\partial s}{\partial y} \cdot \sigma_0 dx dy = 0 \\
 & \lambda_1 \iint x \cdot \left( 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \lambda_2 \iint y \cdot \left( 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_3 \iint x^2 \cdot \left( 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \lambda_4 \iint y^2 \cdot \left( 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_5 \iint \left( 1 + \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy = \iint w dx dy + \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} dx dy \\
 & \lambda_1 \iint x \cdot \left( -\frac{\partial s}{\partial y} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} y \right) \cdot \sigma_0 dx dy + \lambda_2 \iint y \cdot \left( -\frac{\partial s}{\partial y} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} y \right) \cdot \sigma_0 dx dy + \\
 & \lambda_3 \iint x^2 \cdot \left( -\frac{\partial s}{\partial y} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} y \right) \cdot \sigma_0 dx dy + \lambda_4 \iint y^2 \cdot \left( -\frac{\partial s}{\partial y} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} y \right) \cdot \sigma_0 dx dy + \\
 & \lambda_5 \iint \left( -\frac{\partial s}{\partial z} \cdot s - y - \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \frac{\psi}{F_s} y \right) \cdot \sigma_0 dx dy = - \iint w \cdot y_c dx dy - \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot y dx dy \\
 & \lambda_1 \iint x \cdot \left( \frac{\partial s}{\partial x} \cdot s + x - \frac{\psi}{F_s} \cdot s \cdot \frac{\Delta}{\Delta'} + \frac{\psi}{F_s} \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot x \right) \cdot \sigma_0 dx dy + \\
 & \lambda_2 \iint y \cdot \left( \frac{\partial s}{\partial x} \cdot s + x - \frac{\psi}{F_s} \cdot s \frac{\Delta}{\Delta'} + \frac{\psi}{F_s} \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot x \right) \cdot \sigma_0 dx dy + \\
 & \lambda_3 \iint x^2 \cdot \left( \frac{\partial s}{\partial x} \cdot s + x - \frac{\psi}{F_s} \cdot s \frac{\Delta}{\Delta'} + \frac{\psi}{F_s} \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot x \right) \cdot \sigma_0 dx dy + \\
 & \lambda_4 \iint y^2 \cdot \left( \frac{\partial s}{\partial x} \cdot s + x - \frac{\psi}{F_s} \cdot s \frac{\Delta}{\Delta'} + \frac{\psi}{F_s} \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot x \right) \cdot \sigma_0 dx dy + \\
 & \lambda_5 \iint \left( \frac{\partial s}{\partial x} \cdot s + x - \frac{\psi}{F_s} \cdot s \frac{\Delta}{\Delta'} + \frac{\psi}{F_s} \frac{\partial s}{\partial x} \frac{\Delta}{\Delta'} \cdot x \right) \cdot \sigma_0 dx dy = \\
 & - \iint K_c w \cdot z_c dx dy + \iint w \cdot x_c dx dy - \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\Delta}{\Delta'} s dx dy + \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\partial s}{\partial x} \cdot \frac{\Delta}{\Delta'} x dx dy \\
 & \lambda_1 \iint x \cdot \left( -\frac{\partial s}{\partial x} \cdot y + \frac{\partial s}{\partial y} \cdot x + \frac{\Delta}{\Delta'} \cdot y \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_2 \iint y \cdot \left( -\frac{\partial s}{\partial x} \cdot y + \frac{\partial s}{\partial y} \cdot x + \frac{\Delta}{\Delta'} \cdot y \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_3 \iint x^2 \cdot \left( -\frac{\partial s}{\partial x} \cdot y + \frac{\partial s}{\partial y} \cdot x + \frac{\Delta}{\Delta'} \cdot y \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_4 \iint y^2 \cdot \left( -\frac{\partial s}{\partial x} \cdot y + \frac{\partial s}{\partial y} \cdot x + \frac{\Delta}{\Delta'} \cdot y \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy + \\
 & \lambda_5 \iint \left( -\frac{\partial s}{\partial x} \cdot y + \frac{\partial s}{\partial y} \cdot x + \frac{\Delta}{\Delta'} \cdot y \cdot \frac{\psi}{F_s} \right) \cdot \sigma_0 dx dy = \iint K_c w \cdot y_c dx dy + \iint \frac{u \cdot \psi - c}{F_s} \cdot \frac{\Delta}{\Delta'} y dx dy
 \end{aligned} \tag{9}$$

式(9)化简为

$$\left\{ \begin{array}{l} \lambda_1 \left( A_1 + \frac{A'_1}{F_s} \right) + \lambda_2 \left( A_2 + \frac{A'_2}{F_s} \right) + \lambda_3 \left( A_3 + \frac{A'_3}{F_s} \right) + \lambda_4 \left( A_4 + \frac{A'_4}{F_s} \right) + \lambda_5 \left( A_5 + \frac{A'_5}{F_s} \right) = A_6 + \frac{A'_6}{F_s} \\ \lambda_1 \left( B_1 + \frac{B'_1}{F_s} \right) + \lambda_2 \left( B_2 + \frac{B'_2}{F_s} \right) + \lambda_3 \left( B_3 + \frac{B'_3}{F_s} \right) + \lambda_4 \left( B_4 + \frac{B'_4}{F_s} \right) + \lambda_5 \left( B_5 + \frac{B'_5}{F_s} \right) = B_6 + \frac{B'_6}{F_s} \\ \lambda_1 \left( D_1 + \frac{D'_1}{F_s} \right) + \lambda_2 \left( D_2 + \frac{D'_2}{F_s} \right) + \lambda_3 \left( D_3 + \frac{D'_3}{F_s} \right) + \lambda_4 \left( D_4 + \frac{D'_4}{F_s} \right) + \lambda_5 \left( D_5 + \frac{D'_5}{F_s} \right) = D_6 + \frac{D'_6}{F_s} \\ \lambda_1 \left( E_1 + \frac{E'_1}{F_s} \right) + \lambda_2 \left( E_2 + \frac{E'_2}{F_s} \right) + \lambda_3 \left( E_3 + \frac{E'_3}{F_s} \right) + \lambda_4 \left( E_4 + \frac{E'_4}{F_s} \right) + \lambda_5 \left( E_5 + \frac{E'_5}{F_s} \right) = E_6 + \frac{E'_6}{F_s} \\ \lambda_1 \left( G_1 + \frac{G'_1}{F_s} \right) + \lambda_2 \left( G_2 + \frac{G'_2}{F_s} \right) + \lambda_3 \left( G_3 + \frac{G'_3}{F_s} \right) + \lambda_4 \left( G_4 + \frac{G'_4}{F_s} \right) + \lambda_5 \left( G_5 + \frac{G'_5}{F_s} \right) = G_6 + \frac{G'_6}{F_s} \\ \lambda_1 \left( H_1 + \frac{H'_1}{F_s} \right) + \lambda_2 \left( H_2 + \frac{H'_2}{F_s} \right) + \lambda_3 \left( H_3 + \frac{H'_3}{F_s} \right) + \lambda_4 \left( H_4 + \frac{H'_4}{F_s} \right) + \lambda_5 \left( H_5 + \frac{H'_5}{F_s} \right) = H_6 + \frac{H'_6}{F_s} \end{array} \right. \quad (10)$$

式(10)中参数取值可对照式(9), 其中式(10)为含有 6 个变量的非线性方程组, 6 个变量和 6 个非线性方程, 利用非线性方程组的解法, 可以求出三维边坡的安全系数值, 从而判断边坡的稳定性状况。

## 2 滑面初始正应力的假定

初始滑面正应力  $\sigma_0(x, y)$  的假定可以分为 3 种:

- (1) 假定  $\sigma_0$  为条柱自重应力, 即  $\sigma_0^{(1)} = w$ ;
- (2) 假定  $\sigma_0^{(2)} = w \cdot \cos \alpha$ , 相当于瑞典法<sup>[6]</sup>的推广, 即忽略条柱间作用力;
- (3) 可设为  $\sigma_0^{(3)} = 1$ , 此假设相当于简化 Bishop 法<sup>[7]</sup>的推广, 即条柱间作用力为水平

$$\sigma_0^{(3)} = \frac{w - cA \sin \alpha_x / F_s + u \Delta \tan \varphi \sin \alpha_x / F_s}{m_a},$$

$$m_a = \cos r_z \left( 1 + \frac{\sin \alpha_x \tan \phi}{F_s \cos r_z} \right)$$

把  $\sigma_0$  的初始假定代入式(10), 可以得到边坡不同的安全系数值。对于每一种初始正应力的假定, 可求出相应的修正参数  $\lambda_1, \lambda_2, \lambda_3, \lambda_4, \lambda_5$  和安全系数的值。如修正参数为负, 说明最终滑面正应力为负, 故须舍去。对于全为正值的修正参数, 也须计算最终正应力, 如出现负值同样需剔除, 这样我们可以判定滑面初始正应力分布对边坡稳定性是否有影响, 若无影响, 则可方便应用于工程实践。

## 3 工程应用

例 1: 引用的是参考文献[9]的工程案例。其滑坡和物理力学参数如图 3 所示。

例 2: 如图 4 所示, 引用的仍是参考文献[9]的工程案例, 边坡有弱层存在, 其滑坡和物理力学参数如图 3 所示。

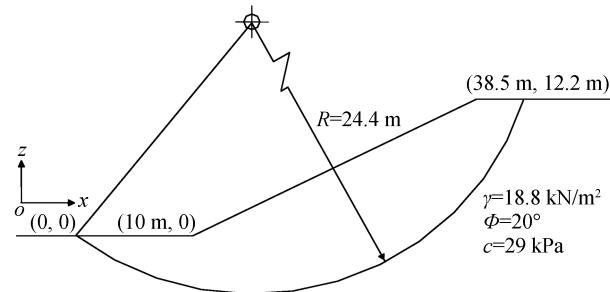


图 3 球形边坡剖面及参数

Fig.3 Cross-section and parameters of a spherical slope

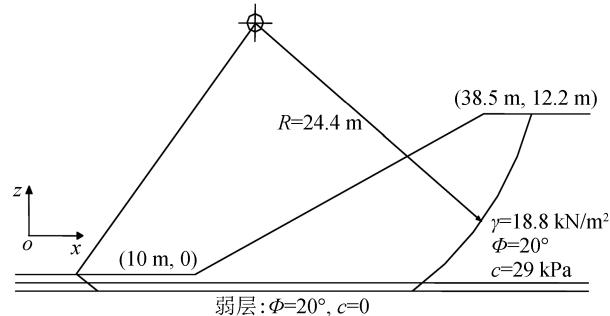


图 4 一般形状边坡主剖面及参数

Fig.4 Cross-section and parameters of a general slope

根据滑面初始正应力的不同假定, 计算工程案例一般形状和球形形状的三维安全系数值, 计算结果列于表 1。

表 1 安全系数比较表

Table 1 Comparison of safety factors

方法	例 1 球形滑体	例 2 一般滑体
二维	2.038	1.383
$\sigma_0^{(1)}$	2.305	1.467
本文方法	2.297	1.538
$\sigma_0^{(3)}$	2.322	1.476
极限平衡解(文献[8])	2.187	1.607
张兴(文献[9])	2.122	1.548
极限分析上限解(文献[10])	2.317	1.754

表1计算结果表明,对于三维球形边坡,不同初始正应力假定,三维安全系数的差别在2.5%以内,与其他方法的最大差别也不超过5%;对于一般三维形状边坡,不同初始正应力假定,三维安全系数的差别在6.2%,与其他方法的差别最大达到28.7%。说明不同初始滑面正应力的假定对球形边坡影响不大,对一般形状边坡影响较大,需要进一步验证滑面正应力分布的合理性才能应用于工程实践。

#### 4 滑面正应力分布的合理性验证

图5为例2的滑面正应力分布图。从图中可以看出,滑面正应力值均为正值,且滑面正应力连续光滑,说明滑面正应力分布是合理的。

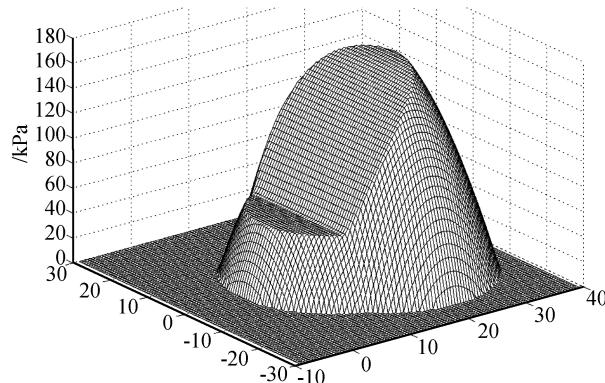


图5 一般形状边坡的三维滑面正应力分布

Fig.5 Normal stress distribution of 3D slip for a slope with general shape

#### 5 结语

假设正应力的分布沿着滑动面,利用严格极限平衡法可以精确求出任意形状滑动面的安全系数。假设三种类型的滑面初始正应力分布,用含有待定参数的修正函数修正滑面初始正应力,根据滑体的三个方向的力和力矩平衡方程,求解出不同初始正应力假定条件下的边坡三维安全系数值。计算结果表明,不同初始正应力假定,对球形三维边坡安全系数影响不大,对一般三维边坡有些影响,但经过滑面正应力验证仍然可以直接应用于滑坡工程。本文方法求出的安全系数值与其他三维极限平衡法求出的

安全系数一致,且有较大的空间效应,满足滑体所有的平衡条件,滑面正应力分布合理,求解结果比较精确。

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